

Micro - 470

## Mechanical Scaling Problem Set

Accelerometer design exercise

# Problem 2: Design an accelerometer

- You have maximum: 1 mm x 1 mm area (for cost reasons)
- Assume you are limited by thermal noise, and want to respond in 5 ms. Design an accelerometer: size of proof mass, and spring constant. Use  $Q=2$
- What displacement would you have at the minimum detectable acceleration?
- Assuming your readout circuit can sense  $\Delta C=5 \text{ aF}$ , what is minimum number of fingers in readout capacitance (assuming single mask fabrication) to read  $a_{\min}$ ? Is this reasonable?
- Hints
  - as always, there is an infinite number of solutions: try to justify your choices from performance (sensitivity), fabrication feasibility, robustness, cost, etc.
  - Choose  $f_{\text{res}}$  to start the design
  - Use linear springs / assume linear behavior
  - How far would mass move at 10 g acceleration?

## Accelerometer sensitivity and thermal noise

$$S_x = \frac{x}{a} = \frac{m}{k} \propto L^2$$

$$S_x = \frac{1}{\omega_0^2}$$

Want low  $f_{\text{res}}$  for high sensitivity, ie high m, low k

If limit chip size, can only play with k (if we use full wafer thickness as the mass)

$$a_{\text{min}} = \sqrt{\frac{4k_b T \omega_0}{m Q}} \sqrt{\Delta f}$$

Want high m, and low  $f_{\text{res}}$  for low thermal noise

But

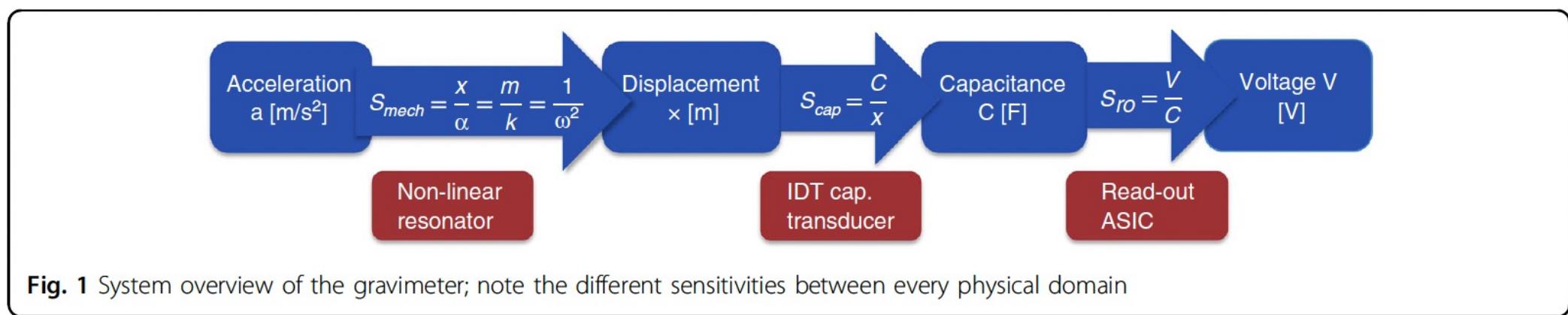
- low  $f_{\text{res}}$  means low bandwidth !
- Readout noise to be considered
- Mechanical robustness...

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# High-resolution MEMS inertial sensor combining large-displacement buckling behaviour with integrated capacitive readout

Brahim El Mansouri<sup>1</sup>, Luke M. Middelburg<sup>1</sup>, René H. Poelma<sup>1</sup>, Guo Qi Zhang<sup>1</sup>, Henk W. van Zeijl<sup>1</sup>, Jia Wei<sup>2</sup>, Hui Jiang<sup>3</sup>,  
Johan G. Vogel<sup>1,4</sup> and Willem D. van Driel<sup>1,4</sup>

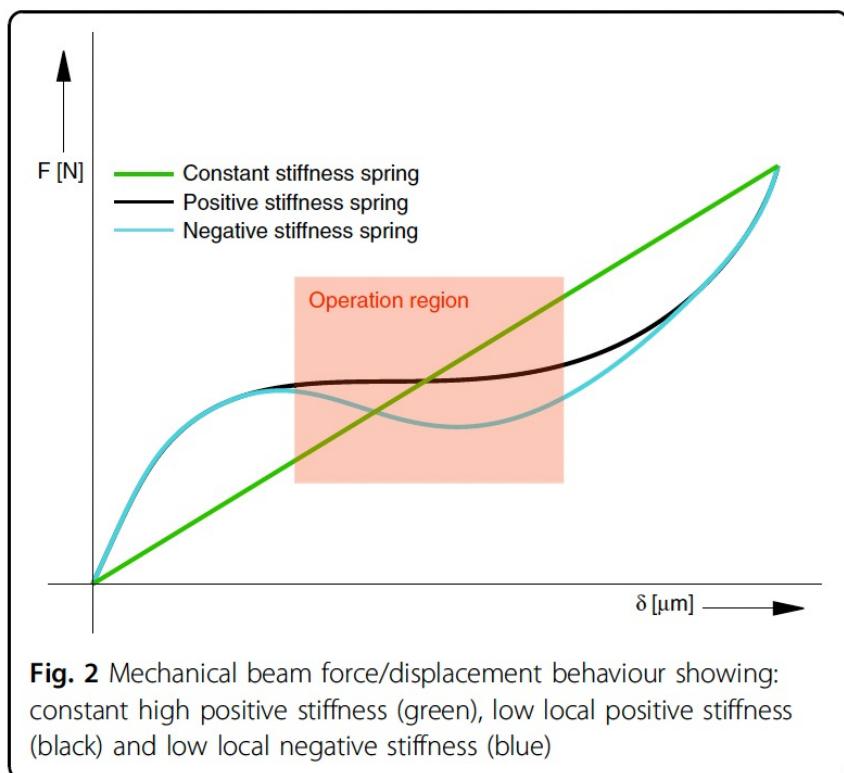


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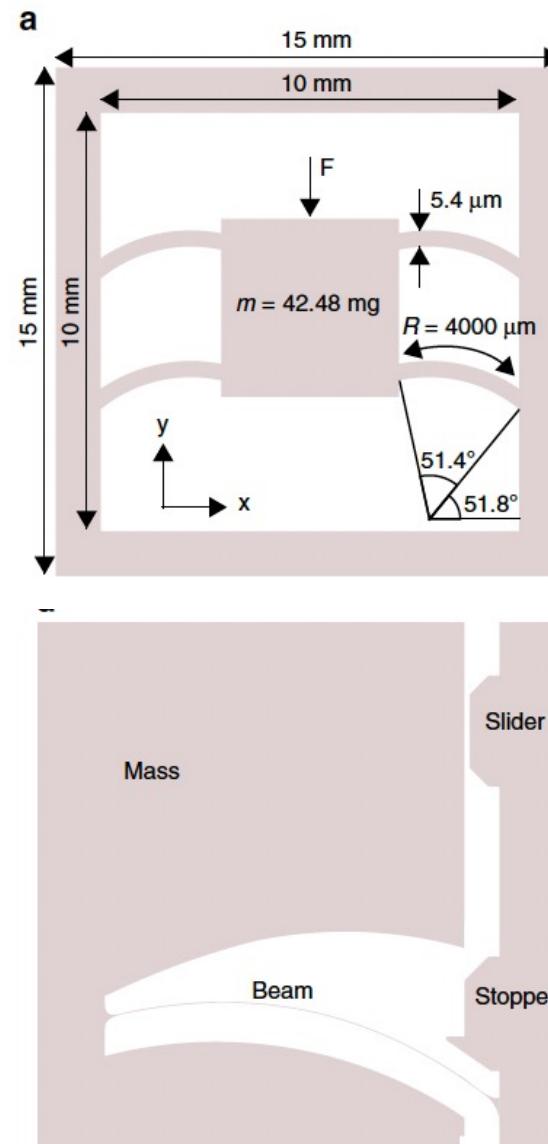
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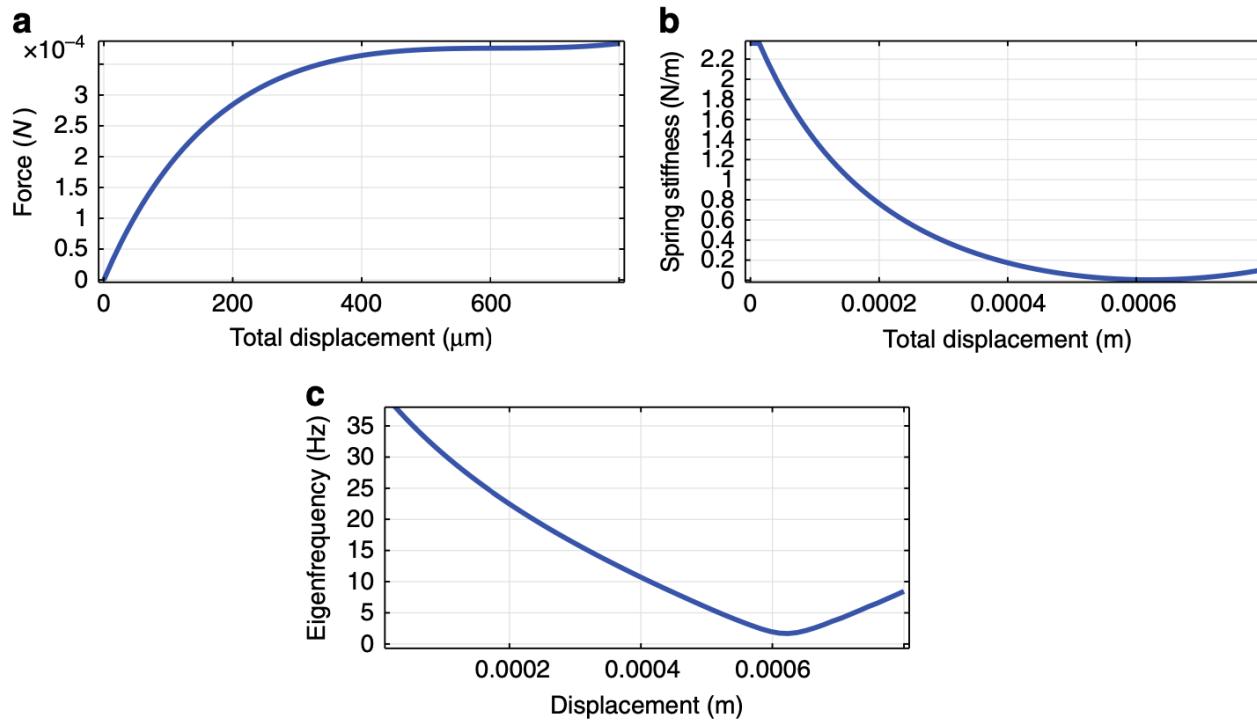
## High-resolution MEMS inertial sensor combining large-displacement buckling behaviour with integrated capacitive readout

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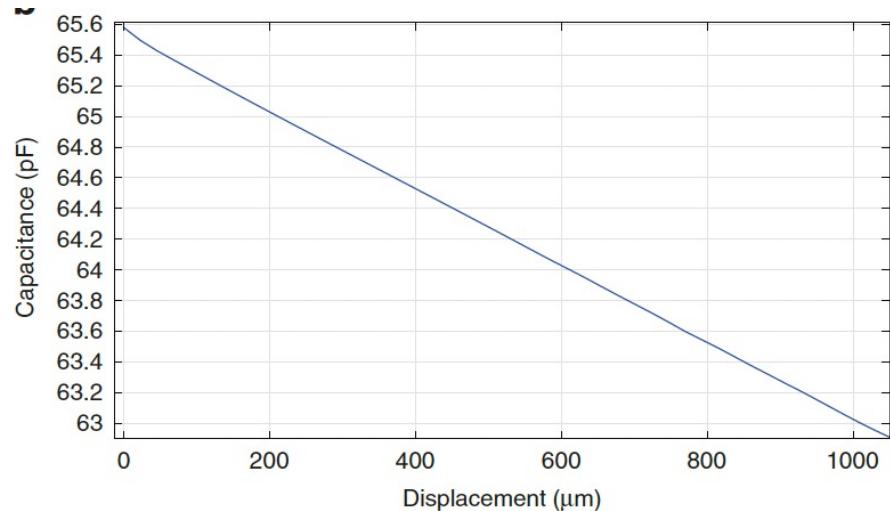
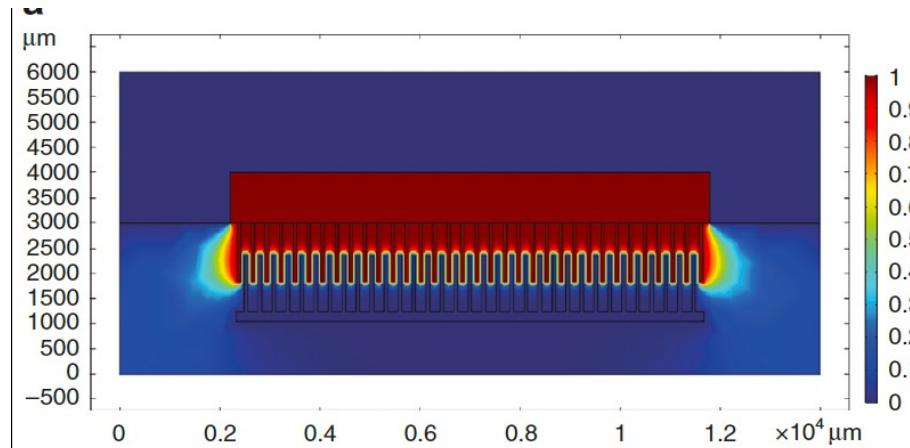


They wanted really “soft” springs to have high sensitivity  
The “trick” used by authors: buckling beams

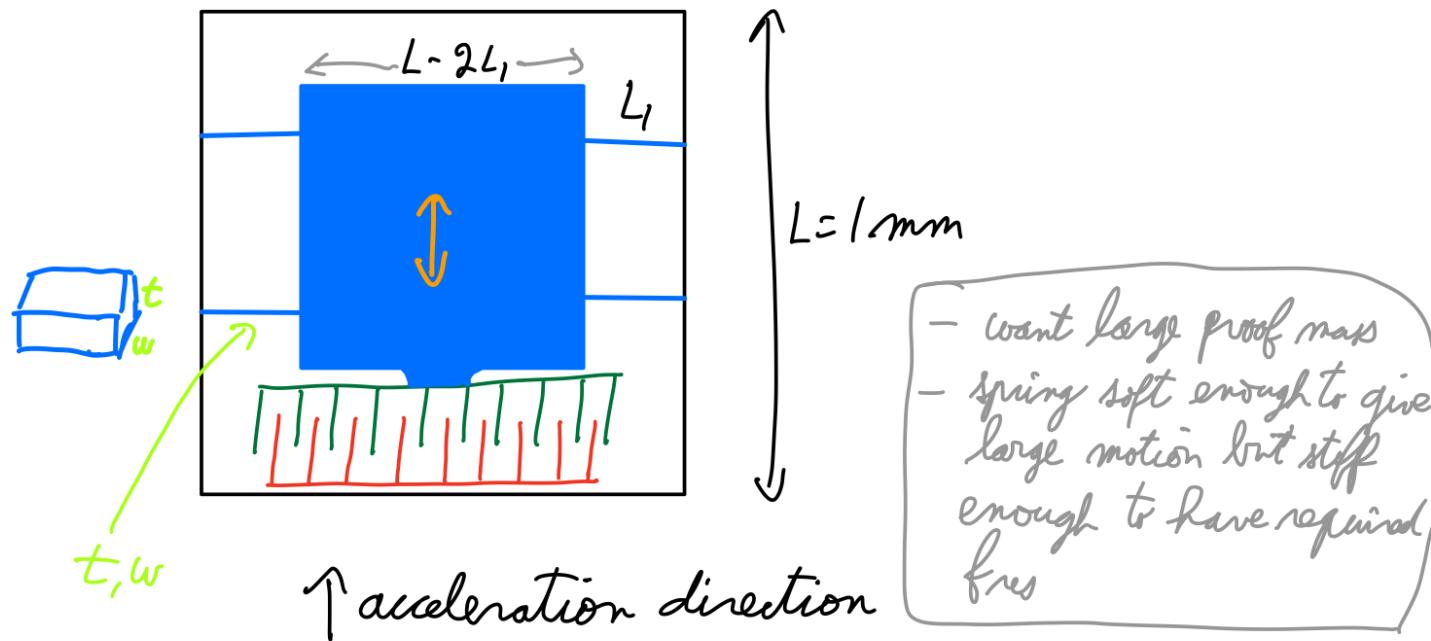




**Fig. 4 FEM simulation results on the non-linear spring design.** **a** Force vs. deflection of the proof mass in the  $y$  direction, **b** stiffness as a function of displacement in the  $y$  direction, **c** resonance (Eigen) frequency as a function of displacement



Readout noise from ASIC, within a BW of 20 Hz, is  
 $0.137 \text{ aF/ } \text{sqrt(Hz)}$



$$1 \text{ mm} \times 1 \text{ mm} \quad 5 \text{ ms response} \quad Q = 2$$

There are many possible solutions!

5 ms response. I chose  $f_{\text{res}} = 500 \text{ Hz}$  to work below resonance

$$\omega_0 = 2\pi f = 2\pi \cdot 500 = 3000 \text{ Hz} = \sqrt{k/m}$$

Choosing  $f_{\text{res}}$  set the sensitivity...

- spring is simple beam, for simplicity

$$k = \frac{1}{4} E \frac{w^3 t}{L_1^3} \text{ per spring. but 4 in parallel}$$

use  $\omega_0 = 500$  Hz to link spring and mass

ie to find link between  $L_1$  and  $w$

$$m = \rho t (L - 2L_1)^2$$

$\rho$  = density of Si  
assume mass is same thickness as the springs (one mass process)

$$\omega_0^2 = \frac{E w^3 t}{L_1^3} \frac{1}{\rho t (L - 2L_1)^2}$$

$$\rightarrow \omega^2 = \frac{\rho \omega_0^2 L_1^3 (L - 2L_1)^2}{E}$$

no  $t$  dependence as  $\frac{k}{m} \propto t$

eg  $L_1 = 250 \mu\text{m}$   $w = 0.8 \mu\text{m}$

- can make  $w$  wider, then get higher  $\omega_0$ , at expense of smaller motion.

→  $k = 0.012 \text{ N/m}$  per spring  
 $= 0.05 \text{ N/m}$  for 4.

Noe can compute  $a_{\min}$

And hence min  $\Delta x$

Choose  $t$  based on process, here assumes 20:1 etch ratio

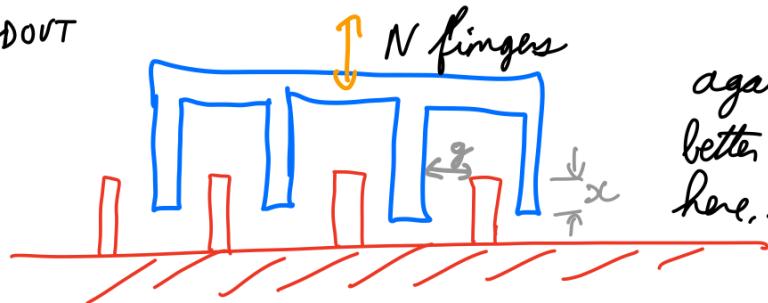
$$a_{\min}^{\text{thermal noise}} = \sqrt{\frac{4h_B T w_0}{Q m}} \sqrt{\Delta f} \quad \Delta f = 200 \text{ Hz}$$

$$\text{for } t = 20 \mu\text{m} \quad a_{\min} = 6.6 \cdot 10^{-4} \text{ m s}^{-2} \quad m = 1.2 \cdot 10^{-8} \text{ kg}$$
$$T = 300 \text{ K} \quad = 66 \mu\text{G}$$

$$\Delta x \text{ for } a_{\min} : \Delta x = \frac{ma}{\hbar} = \frac{1.2 \cdot 10^{-8} \cdot 6.6 \cdot 10^{-4}}{0.05} = 1.5 \cdot 10^{-10} \text{ m}$$
$$\Delta x_{\min} = \underline{0.15 \text{ nm}}$$

- need to detect 0.15 nm
- but also e.g 1 G  $\rightarrow \Delta x = \frac{1.2 \cdot 10^{-8} \cdot 9.8}{0.05} = 2.3 \mu\text{m}$
- would like thicker  $t$  for higher mass, but limited by etch process to  $t \sim 20$ .

READOUT



again, many options  
better could be differential  
here, simpler solution.

$$C = \epsilon_0 \frac{t x}{g}$$

$$\Delta C = \epsilon_0 t \frac{\Delta x}{g}$$

per finger

$$\Delta C_{\min} = \frac{\epsilon_0 t}{g} \Delta x_{\min} \cdot N$$

$$g = 1 \mu\text{m} \quad t = 20 \mu\text{m}$$

$$\Delta C_{\min} = 3 \cdot 10^{-20} \text{ F per finger}$$

want  $g$  as small as  
possible.  
but difficult to make  
 $g < t/20$

$$10^{-10} \cdot 20 \cdot 1.5 \cdot 10^{-10}$$

$$3 \cdot 10^{-20}$$

readout circuit can sense  $5 \text{ aF} = 5 \cdot 10^{-18} \text{ F}$

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$$N \text{ fingers} = \frac{5 \cdot 10^{-18}}{3 \cdot 10^{-20}} = 167 \text{ fingers}$$

Was it fair to ignore the mass of the comb finger?